

**Stretching, bending, twisting  
and coiling: how to build a  
fluid-mechanical sewing machine**

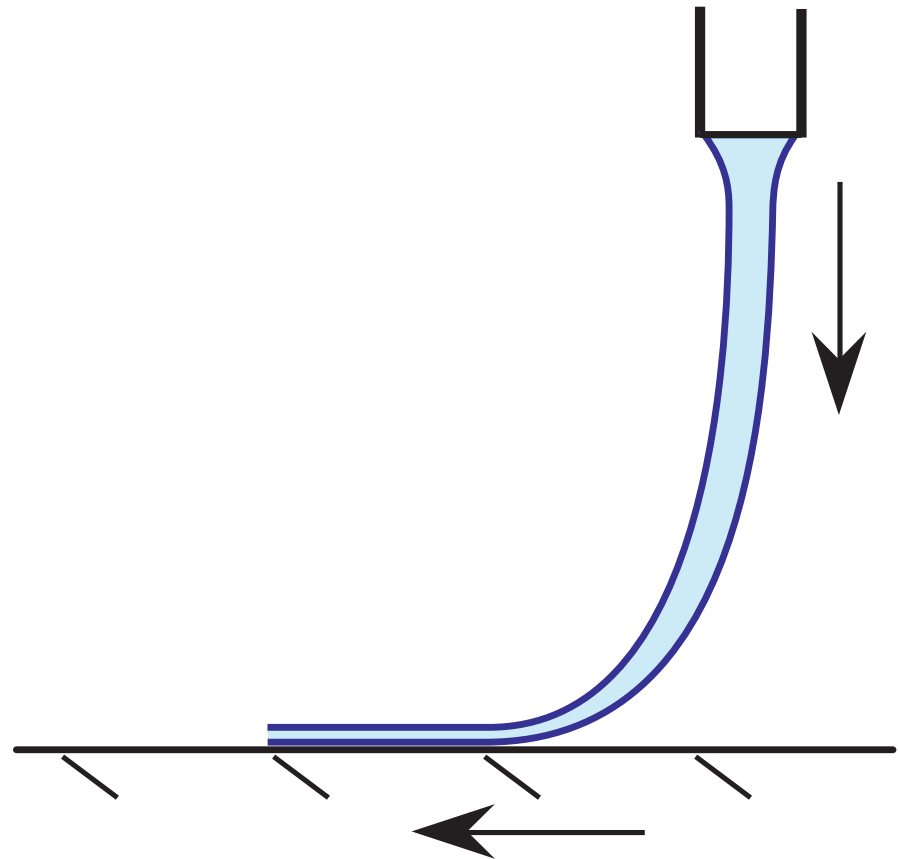
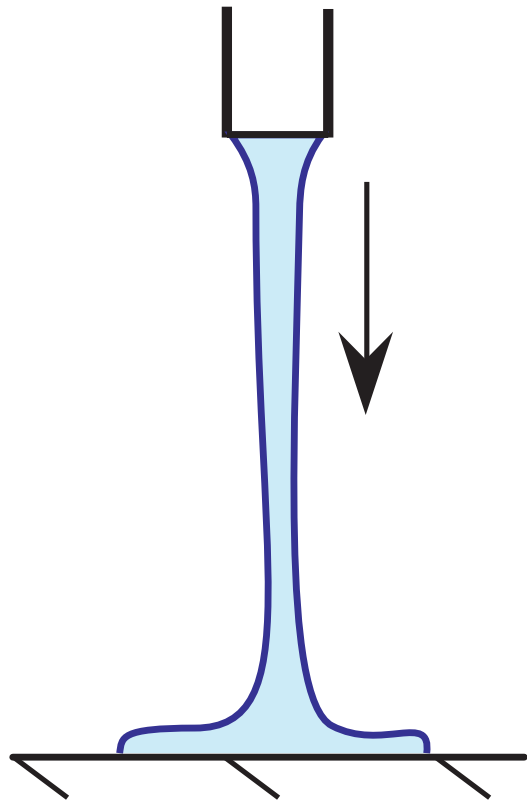
**John Lister**

**ITG, DAMTP**

**Trinity Mathematical Society, 6 February 2012**

# Something Simple for Breakfast?

A thread of viscous fluid falling on a surface

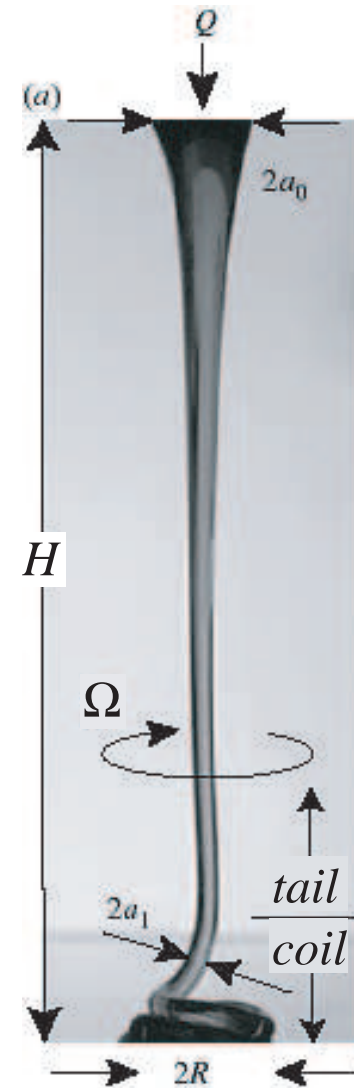


# Coiling on a Stationary Surface

- $\exists$  Steady coiling solution (helical wave)



Mahadevan et al. 2000



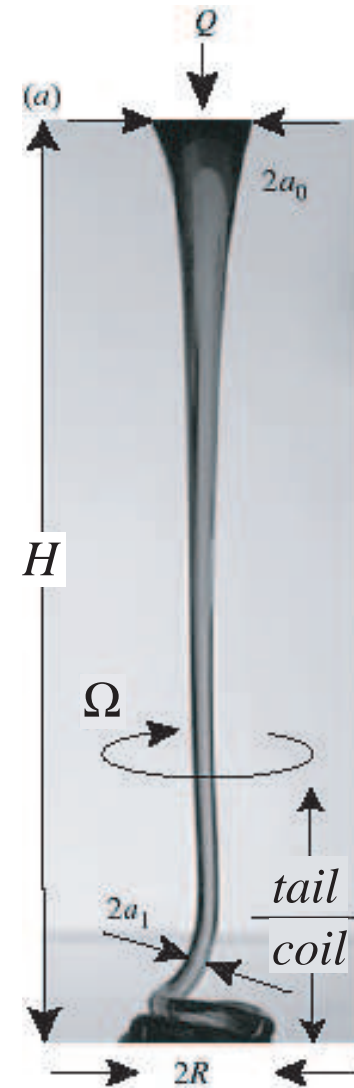
Ribe 2004  $\Leftarrow \Leftarrow$

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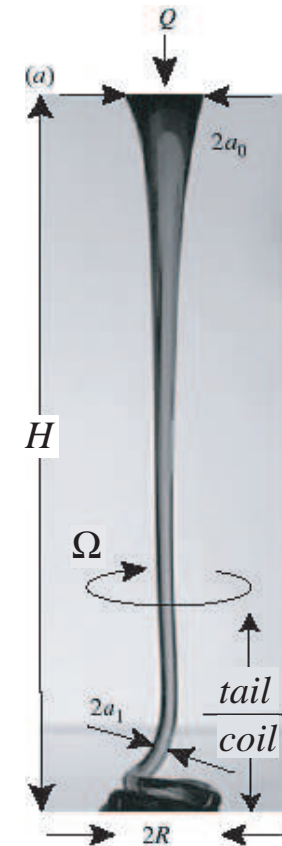
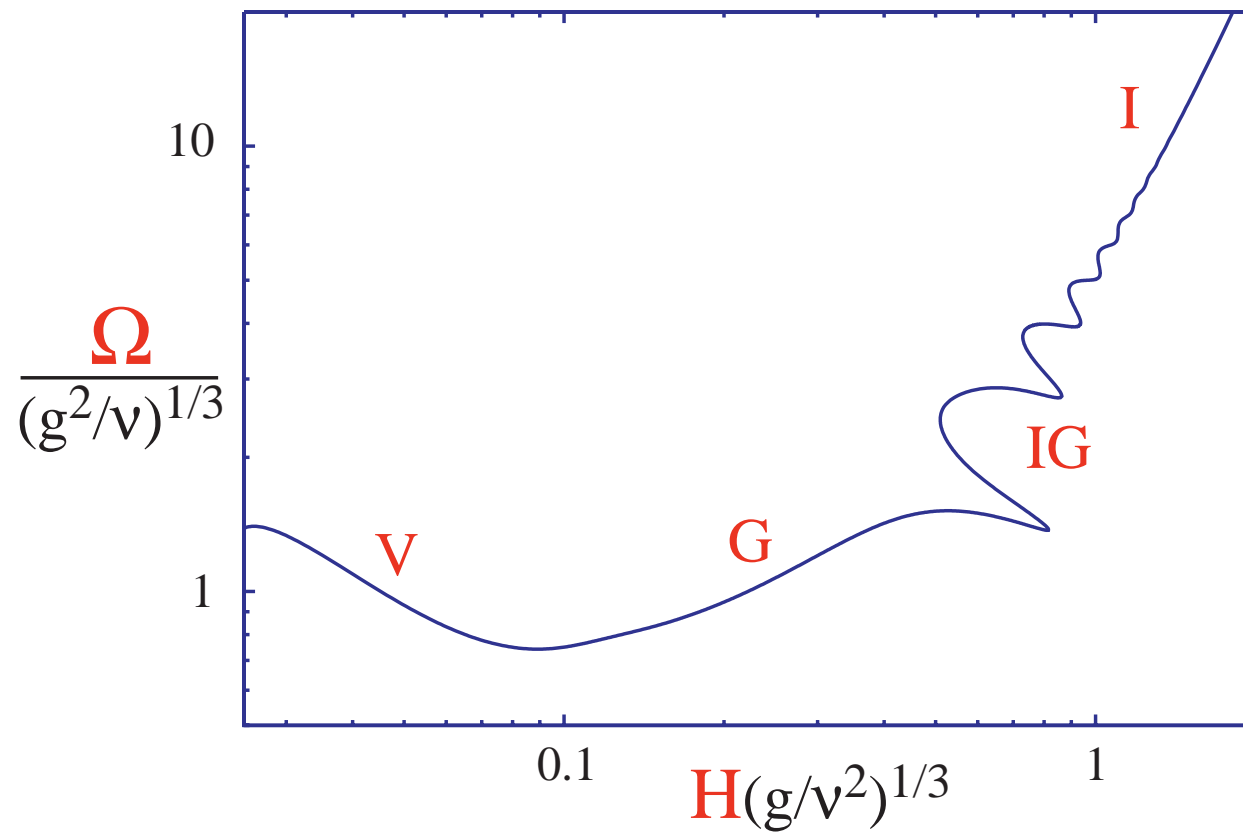


Mahadevan et al. 2000



Ribe 2004  $\Leftarrow \Leftarrow$

# Not So Simple!



What determines the frequency?

What happens if the surface is moving?

## On Board

Stress linear in velocity gradients. Translation and rotation irrelevant

Viscous stress in simple shear

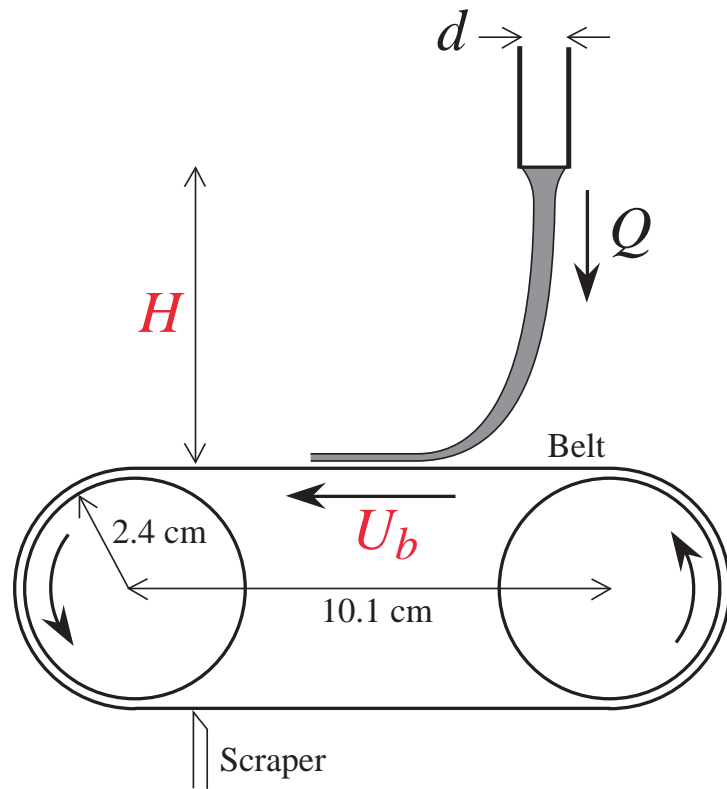
Viscous stress in axial extension

Mass conservation for a thread

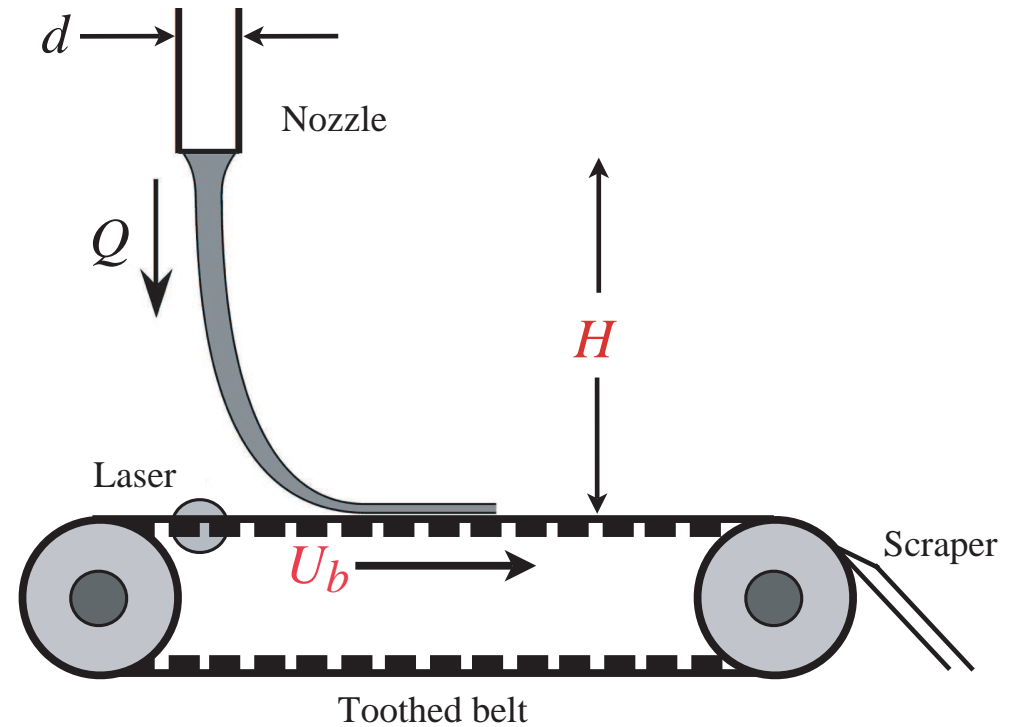
Steady vertical fall solution - infinite fall, finite fall.

Bending

# Fall onto a Moving Belt



Chiu-Webster & Lister 2006



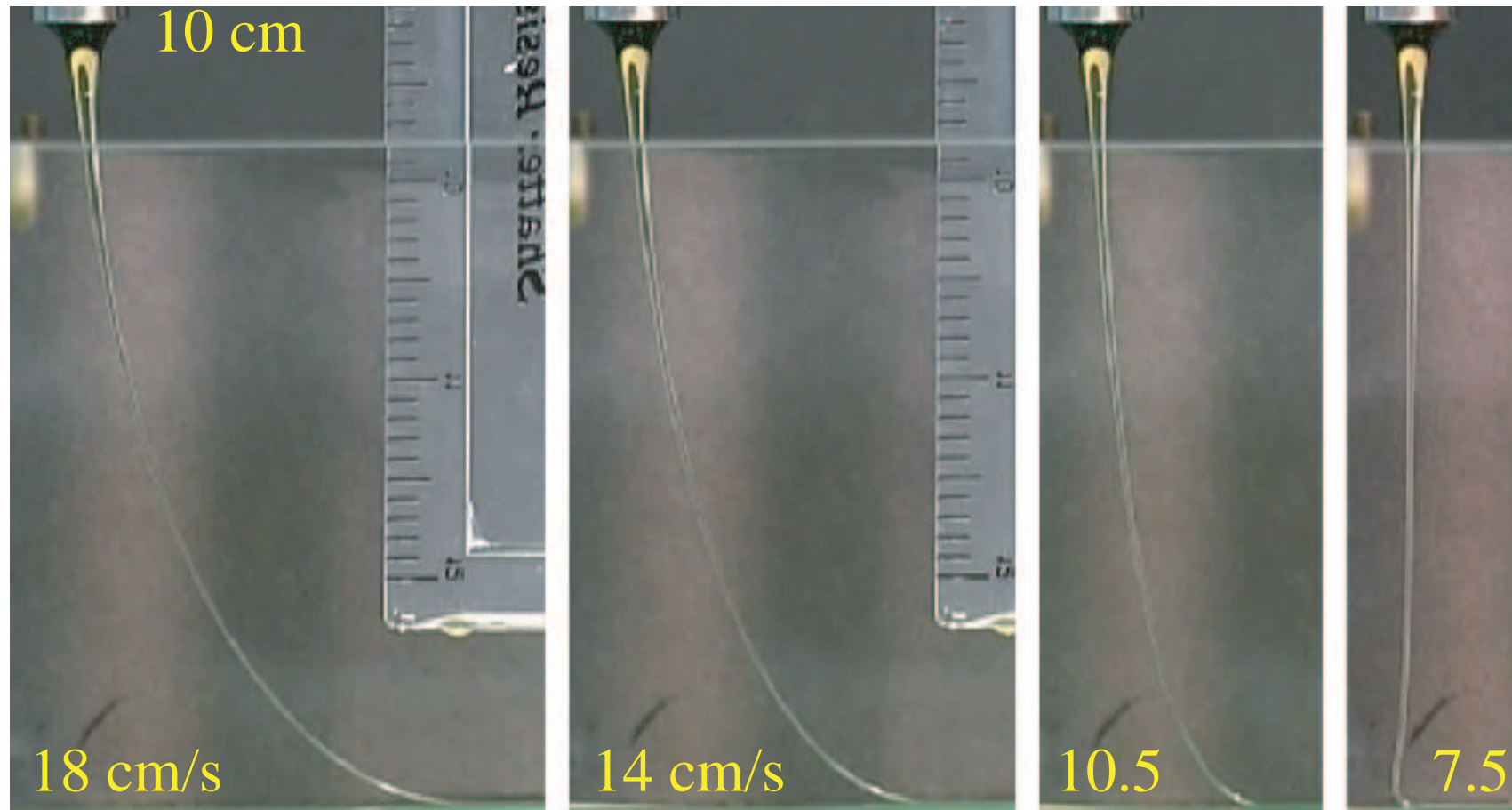
Morris, Ribe, Dawes & Lister 2008

- Belt speed  $U_b$  and fall height  $H$  are the main control parameters
- Let  $U_f$  be the ‘free-fall’ speed
  - speed if  $T = 0$  after vertical fall through  $H$

# Steady Stretching Catenary Shapes for $U_b \gtrsim U_f$

Experiments with Golden Syrup

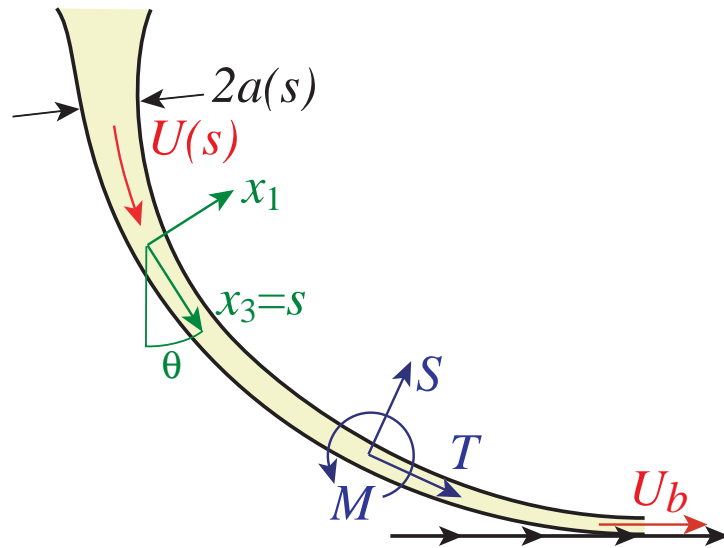
Chiu-Webster & Lister 2006



Thread is in tension – catenaries with variable mass per unit length



# Equations for Steady Catenaries (with bending)



Model by centreline variables

$$T = \int \sigma_{33} dA \quad S = \int \sigma_{13} dA$$

$$M = \int x_1 \sigma_{33} dA$$

Rolling conditions on belt

Clamped conditions at nozzle

Geometry  $\theta' = \kappa$ ,  $x' = \sin \theta$ ,  $z' = -\cos \theta$

Mass  $a^2 U = \text{const.}$

Vertical force balance  $(T \cos \theta - S \sin \theta)' = \pi a^2 \rho g$

Horizontal force balance  $(T \sin \theta + S \cos \theta)' = 0$

Moment balance  $M' = S$

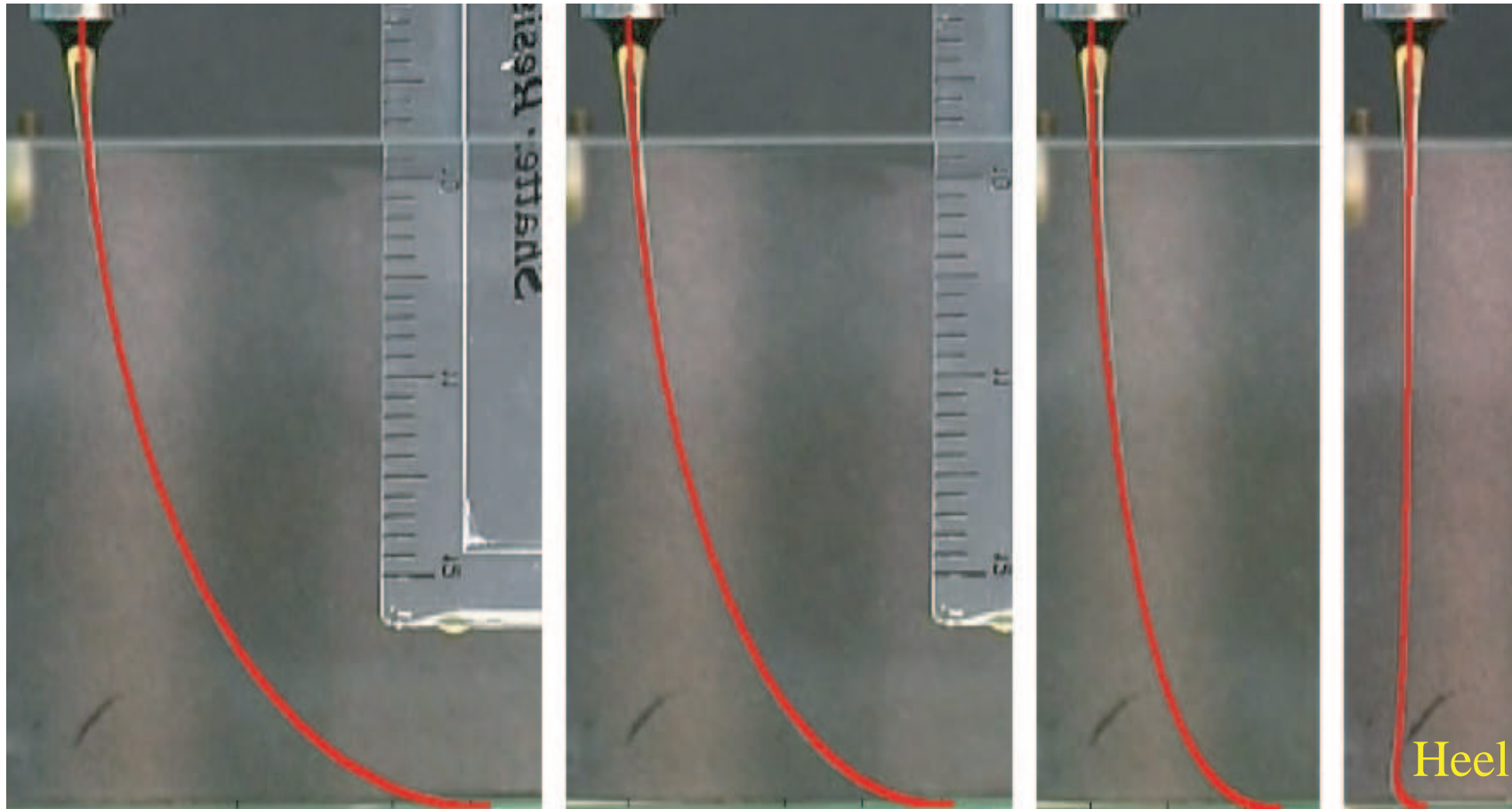
Resistance to stretching  $T = 3\mu\pi a^2 U'$

Resistance to bending  $M = \frac{3}{4}\mu\pi a^4 U \kappa'$

8th-order ODEs

# Steady Stretching Catenary Shapes for $U_b \gtrsim U_f$

Very good agreement between simple theory and experiments



As  $U_b \searrow U_f$  the shape develops a heel (to be continued ...)

## Motion in 3D for Coiling Etc. (no inertia for simplicity)

Shape  $\mathbf{r}(s, t)$       Tangent  $\mathbf{d}_3 = \mathbf{r}'$  and material  $\mathbf{d}_1, \mathbf{d}_2$

Kinematics  $\frac{D\mathbf{r}}{Dt} = \mathbf{U}, \quad \frac{D\mathbf{d}_i}{Dt} = \boldsymbol{\omega} \wedge \mathbf{d}_i, \quad \frac{D}{Dt} = \frac{\partial}{\partial t} + U_s \frac{\partial}{\partial s}$

Mass conservation  $\frac{Da^2}{Dt} = -(\mathbf{U}')_3 a^2$

Forces & moments  $\mathbf{T}' + \mathbf{F} = \mathbf{0}, \quad \mathbf{M}' + \mathbf{d}_3 \wedge \mathbf{T} + \mathbf{G} = \mathbf{0}$

Constitutive eqns  $\mathbf{M} = \frac{1}{4}\pi a^4 \mu (3\mathbf{I} - \mathbf{d}_3 \mathbf{d}_3) \cdot \boldsymbol{\omega}' \quad T_3 = 3\pi a^2 \mu (\mathbf{U}')_3$

for the two bending directions, twisting and stretching

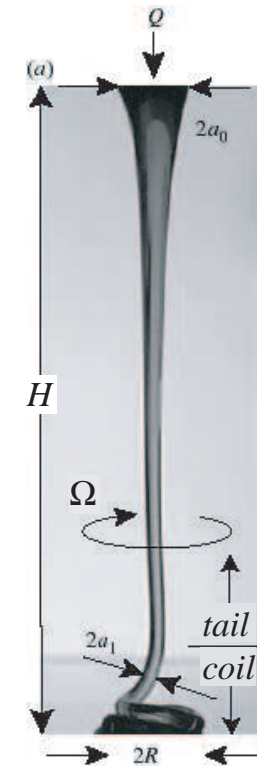
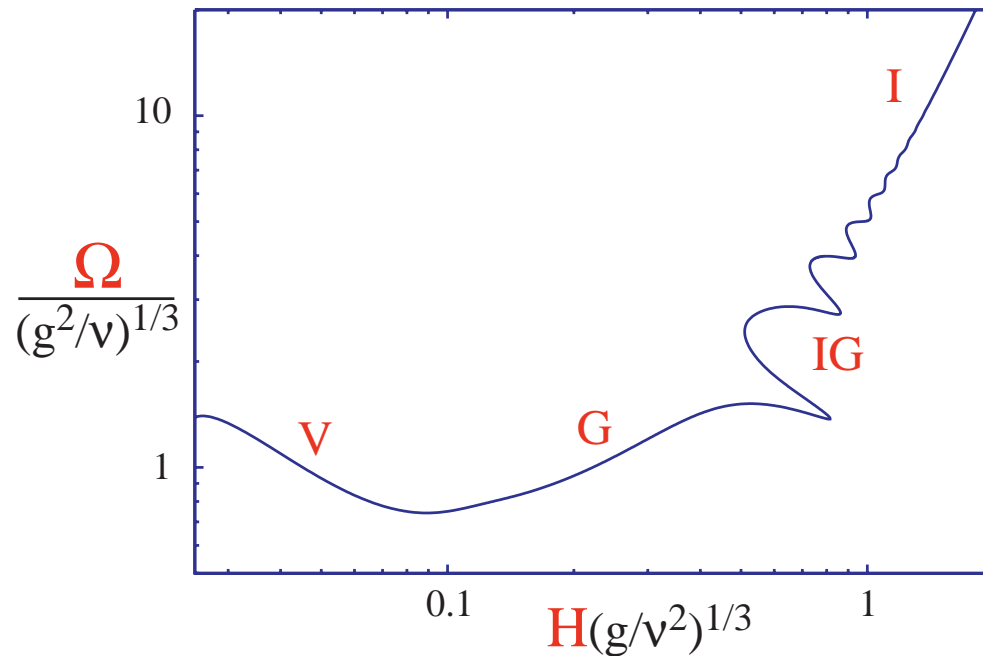
- System for velocities and angular velocities is a 12th order PDE  
The kinematics of position, angle, radius and arclength  
adds another 8 (or 9) orders!

# The Four Regimes of Steady Coiling

Ribe et al. 2006

Numerical solutions show four regimes.

Backed by experiments. Explained by scaling arguments.



- Viscous: global buckling (like toothpaste)
- Gravitational: balance gravity and bending stresses in coil
- Inertial: balance inertia and bending stresses in coil
- Inertio-Gravitational: (see next slide)

# Oscillations of a Hanging String

Vertical hanging string, uniform radius  $a$ , negligible stiffness

Vertical force balance  $T' = \pi a^2 \rho g \Rightarrow$  linear variation of tension

Small horizontal displacements  $(Tx')' = \pi a^2 \rho \ddot{x}$

Seek eigenmodes  $x = X(s)e^{i\Omega t}$

$$\frac{d}{ds} \left( s \frac{dX}{ds} \right) = -\frac{\Omega^2}{g} X, \quad X(H) = 0, \quad sX' \rightarrow 0 \text{ at } 0$$

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Solution  $X = J_0(2\Omega\sqrt{s/g})$  with eigenfrequencies  $\Omega = \frac{j_{0,n}}{2} \sqrt{\frac{g}{H}}$

Frequencies modified by nonuniform radius

and by Coriolis effects if there is velocity  $U$  along string

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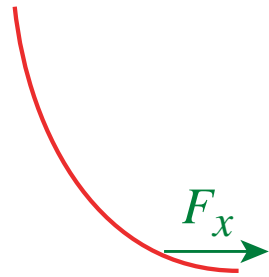
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**IG regime** is due to **resonance** of coiling with the pendulum modes

# Back to the Dragged Thread

- There are three basic regimes (and steady solutions in each):

Dragged catenary



$$U_b > U_f$$

Transitional Heel



$$U_b \approx U_f$$

Compressional Heel



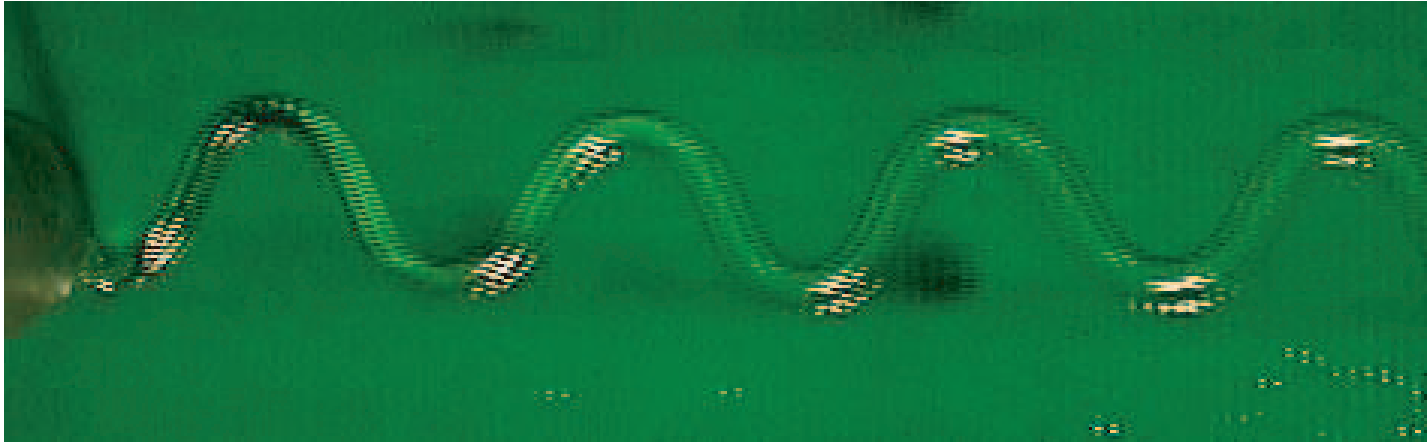
$$U_b < U_f$$

- Expect compressional heel to be unstable to buckling
- Linear perturbations to the planar basic state decouple into
  - a 12th order out-of-plane system (bending & twisting)
  - a 9th order in-plane system (bending & stretching)
- At leading order, transitional heel becomes unstable not when  $F_z = 0$  but when  $F_x = 0$ . Not buckling but overbalancing!



# Onset of Oscillations for $U_b = U_f - O(\epsilon^{1/2})$

Thread becomes unstable to sinusoidal meanders



$$H = 8.5 \text{ cm}$$

$$U_b = 4.0 \text{ cm/s}$$



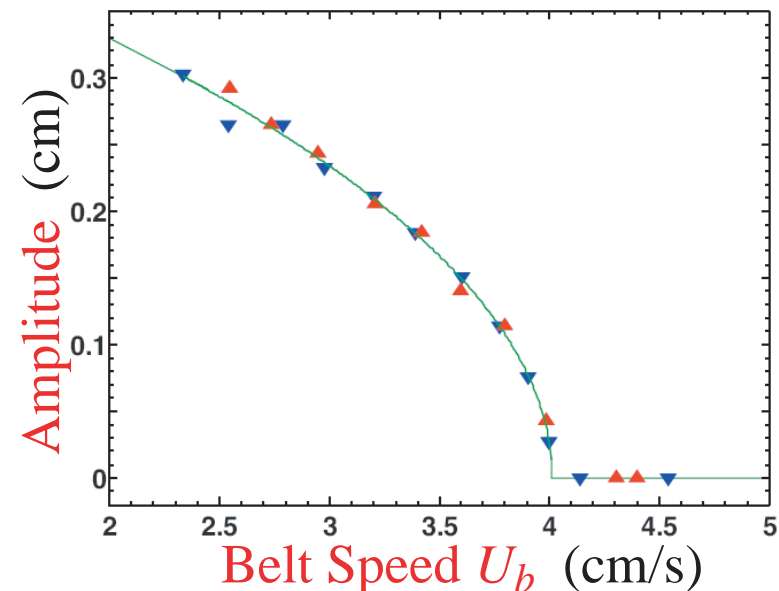
Onset is a Hopf bifurcation

Onset point and frequency can be calculated

numerically (Ribe, Lister & C-W 2006)

asymptotically (Blount & Lister 2011)

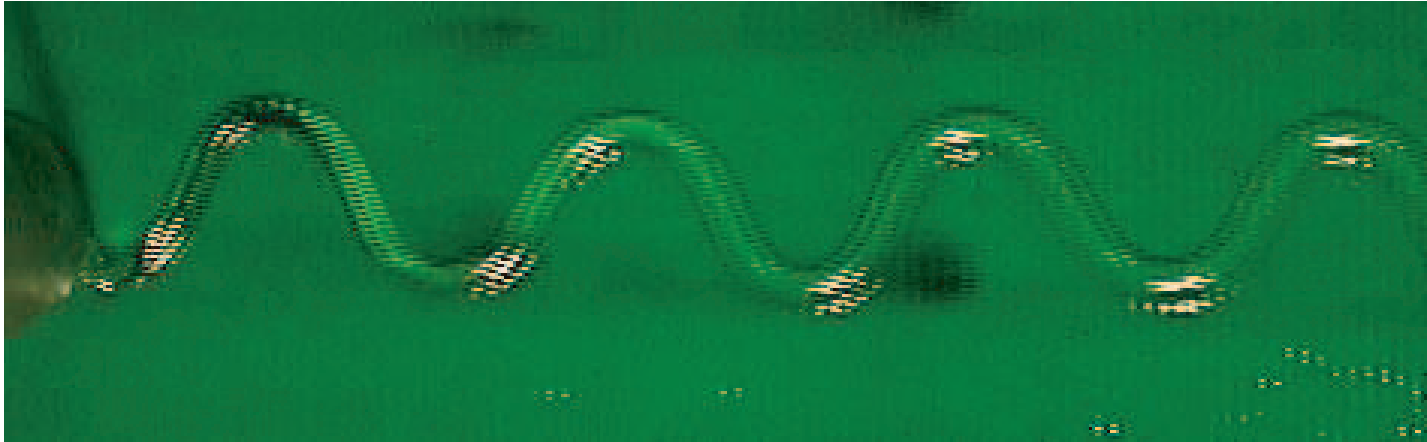
$$\text{e.g. } \Omega \sim \frac{U_f}{H} \frac{2.53}{(\epsilon \ln \epsilon)^{1/2}} \text{ for } \epsilon = \frac{a_f}{H} \ll 1$$



Morris et al. 2008

# Onset of Oscillations for $U_b = U_f - O(\epsilon^{1/2})$

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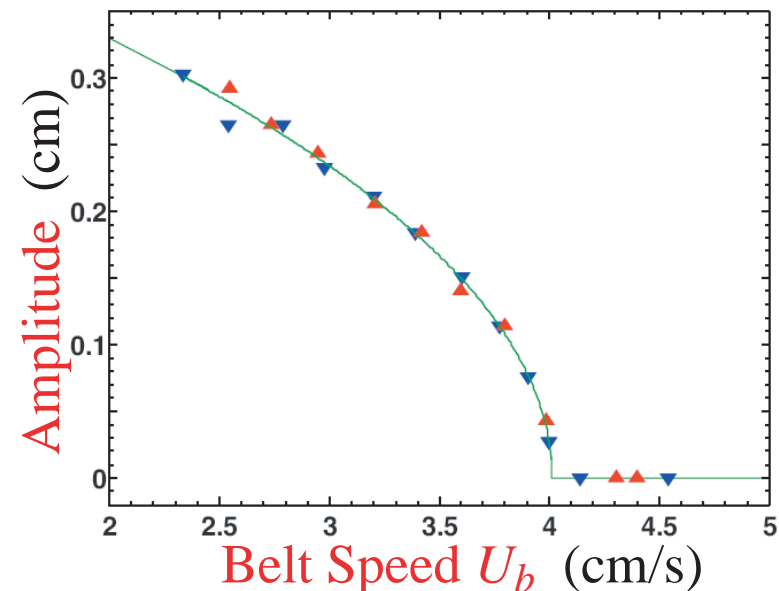
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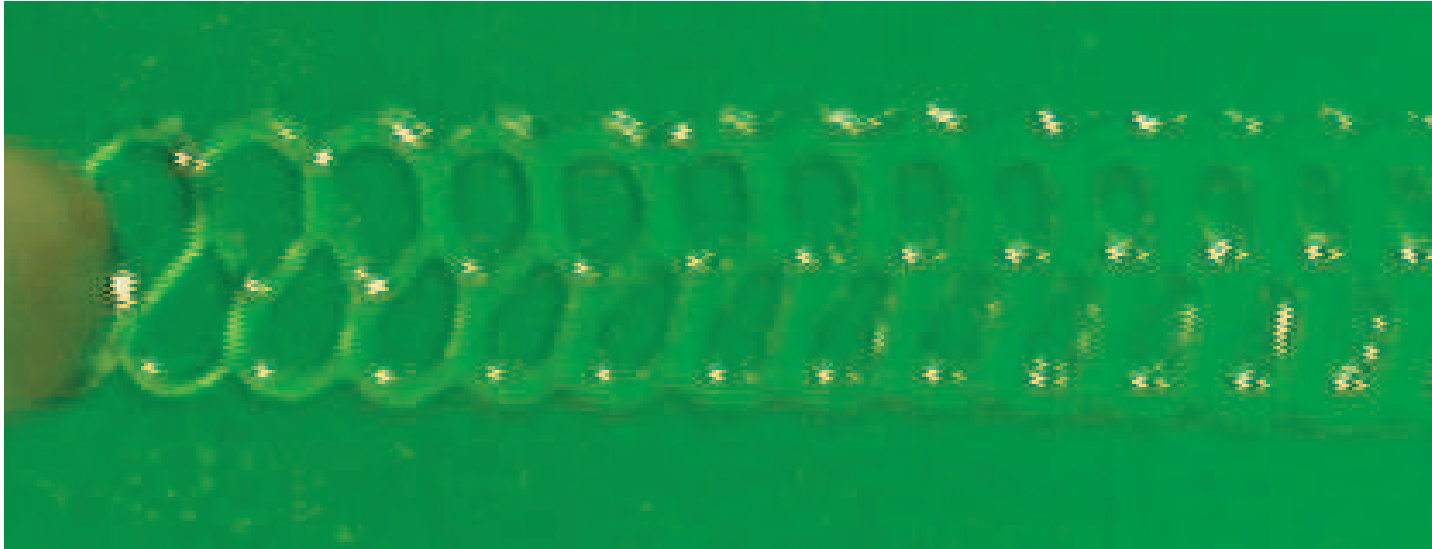
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# Further Bifurcations for $U_b \lesssim U_f$

Braiding

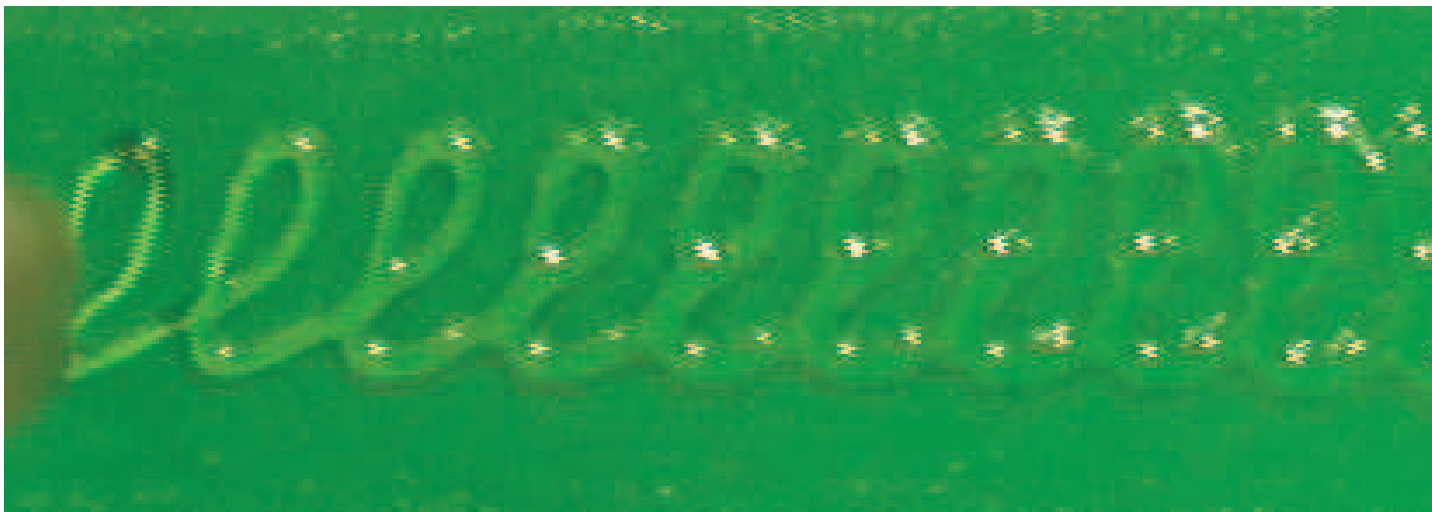


$$H = 8.25 \text{ cm}$$

$$U_b = 1.1 \text{ cm/s}$$



Slanted loops



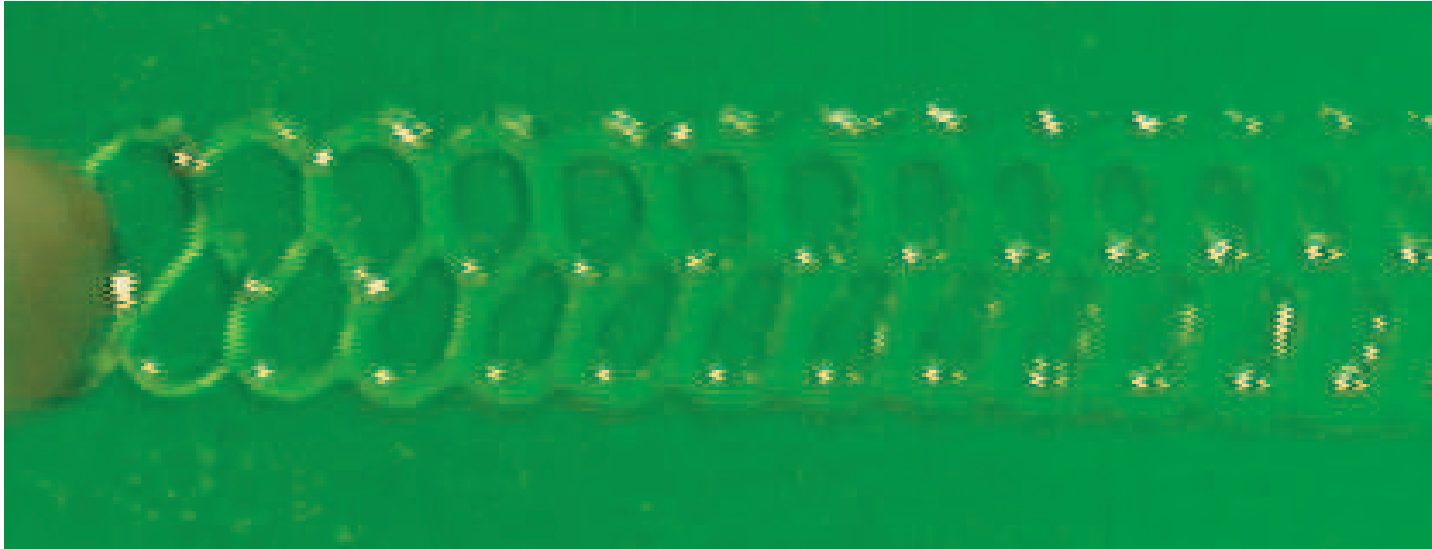
$$H = 8.0 \text{ cm}$$

$$U_b = 1.2 \text{ cm/s}$$



# Further Bifurcations for $U_b \lesssim U_f$

Braiding

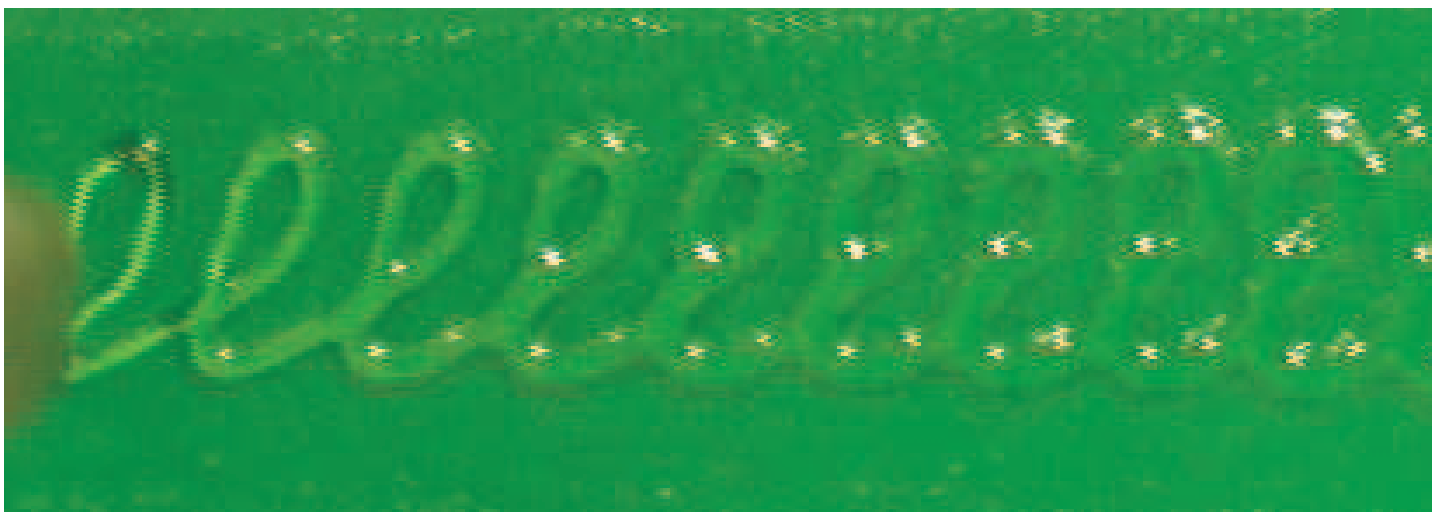


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Slanted loops



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## Further Bifurcations for $U_b \lesssim U_f$

Coiling – the obvious result as  $U_b \rightarrow 0$

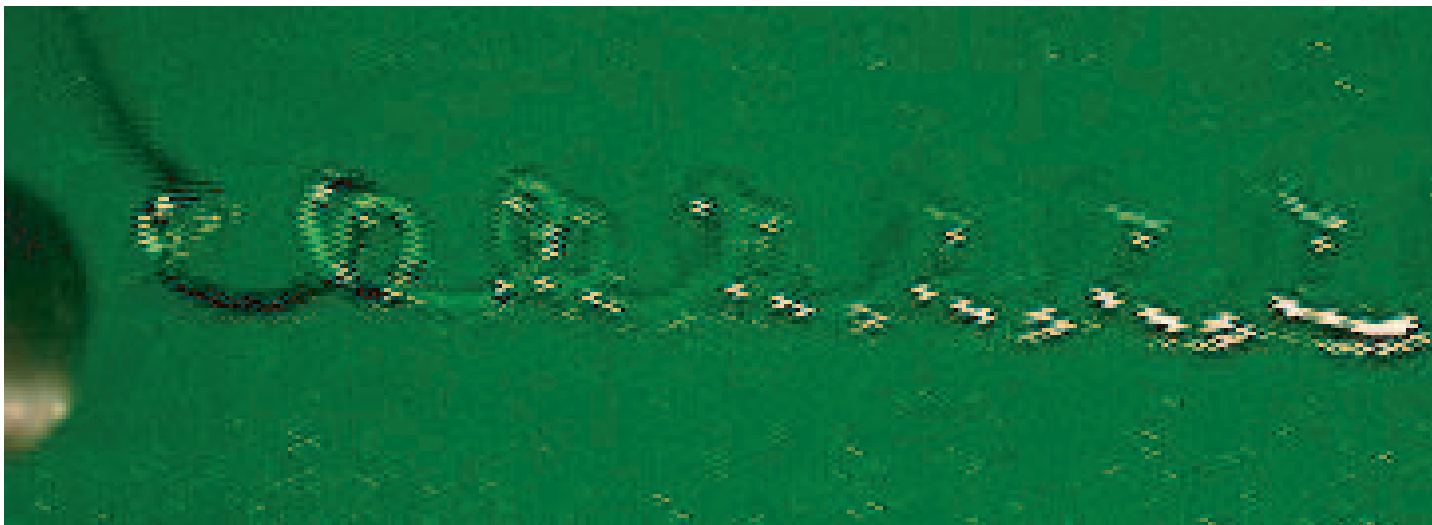


$$H = 8.5 \text{ cm}$$

$$U_b = 4.0 \text{ cm/s}$$



Double coiling



$$H = 8.5 \text{ cm}$$

$$U_b = 1.8 \text{ cm/s}$$



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Coiling – the obvious result as  $U_b \rightarrow 0$

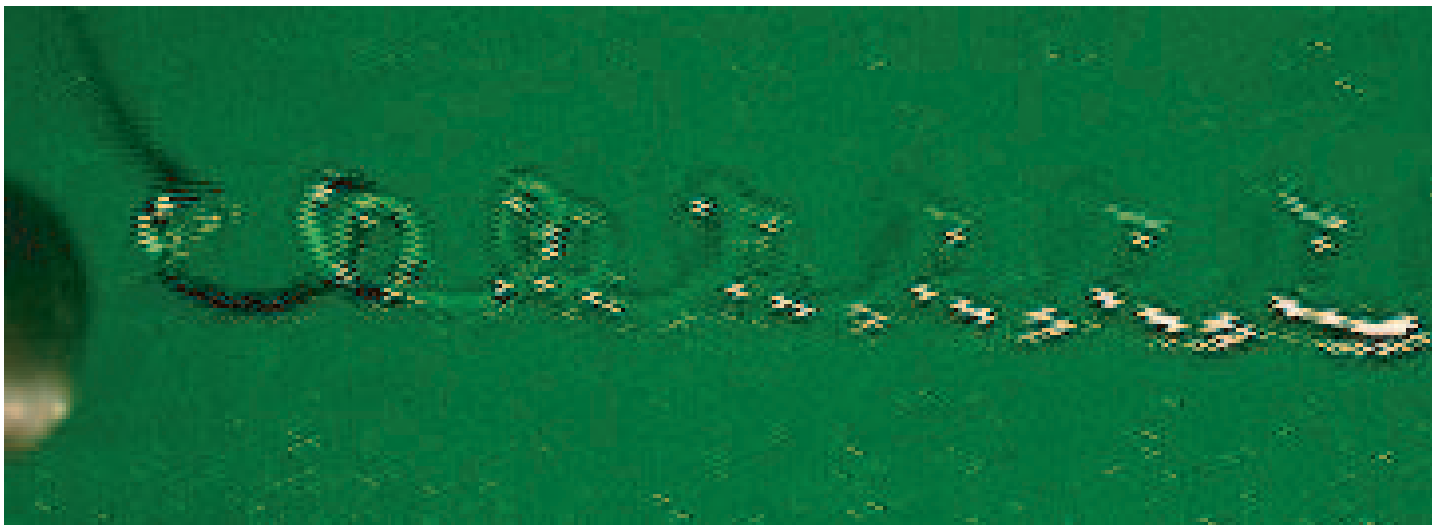


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Double coiling



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# Further Bifurcations for $U_b \lesssim U_f$

Figure-of-eight



$$H = 9.5 \text{ cm}$$

$$U_b = 3.3 \text{ cm/s}$$



W



$$H = 8.5 \text{ cm}$$

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# Further Bifurcations for $U_b \lesssim U_f$

Figure-of-eight



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W



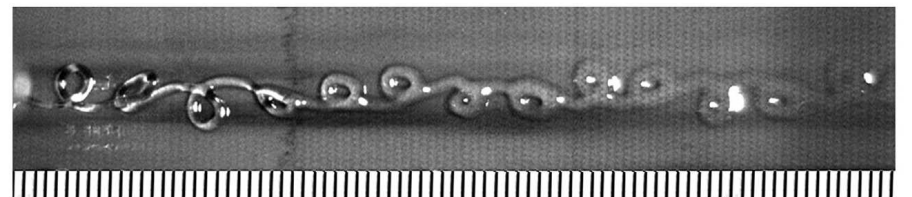
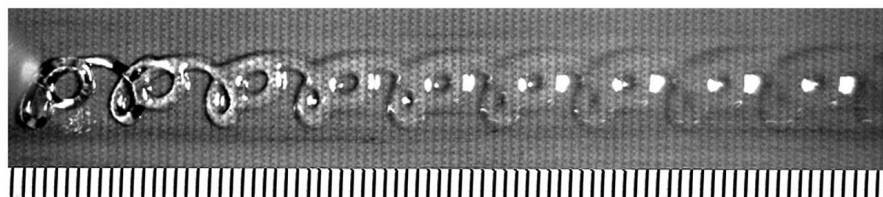
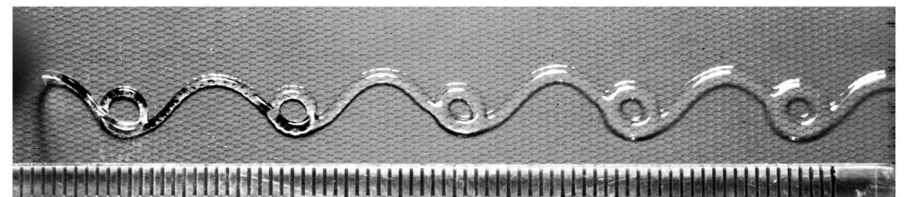
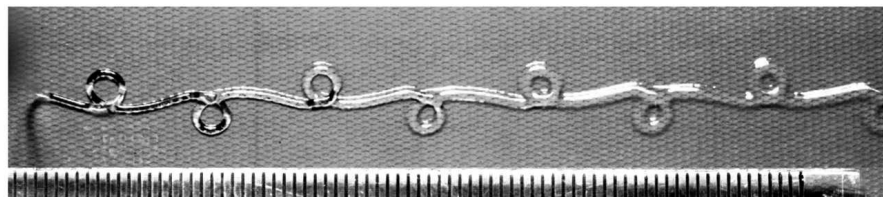
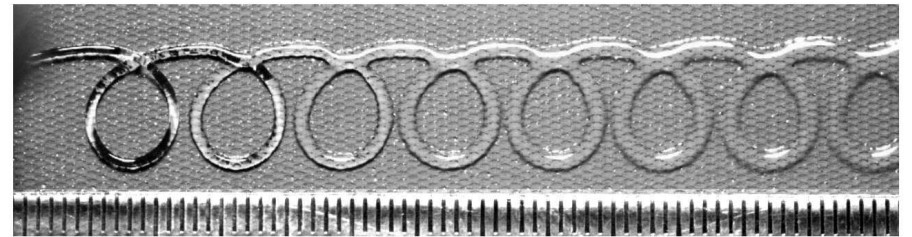
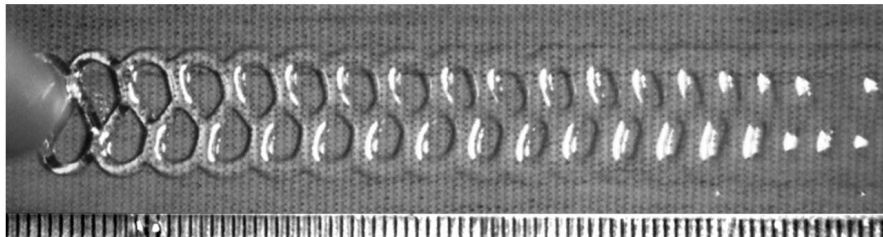
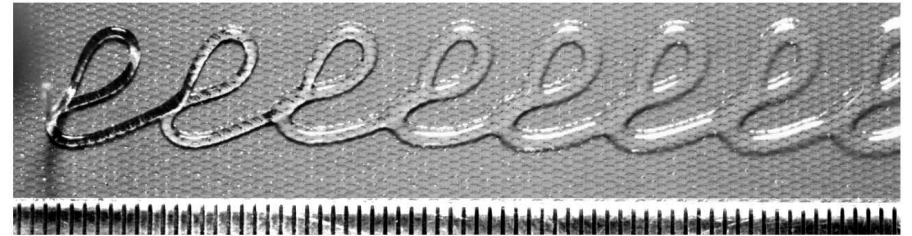
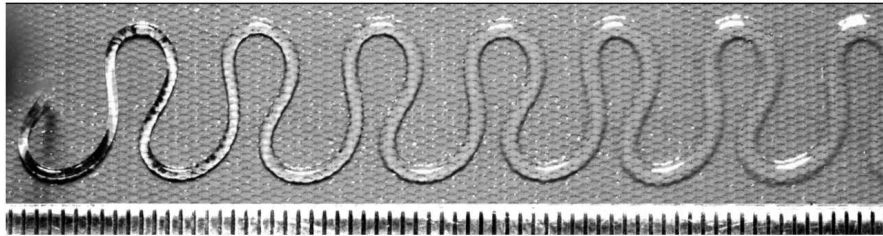
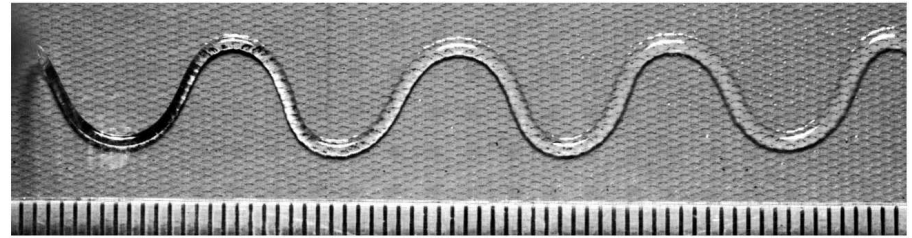
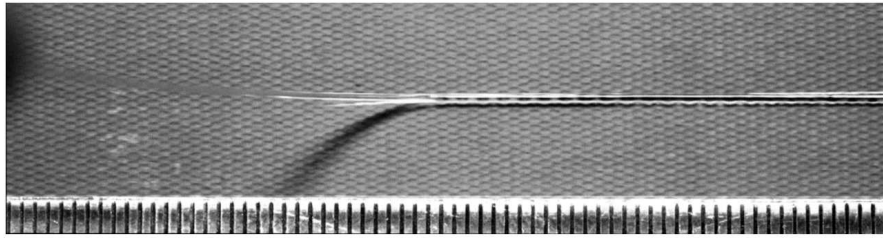
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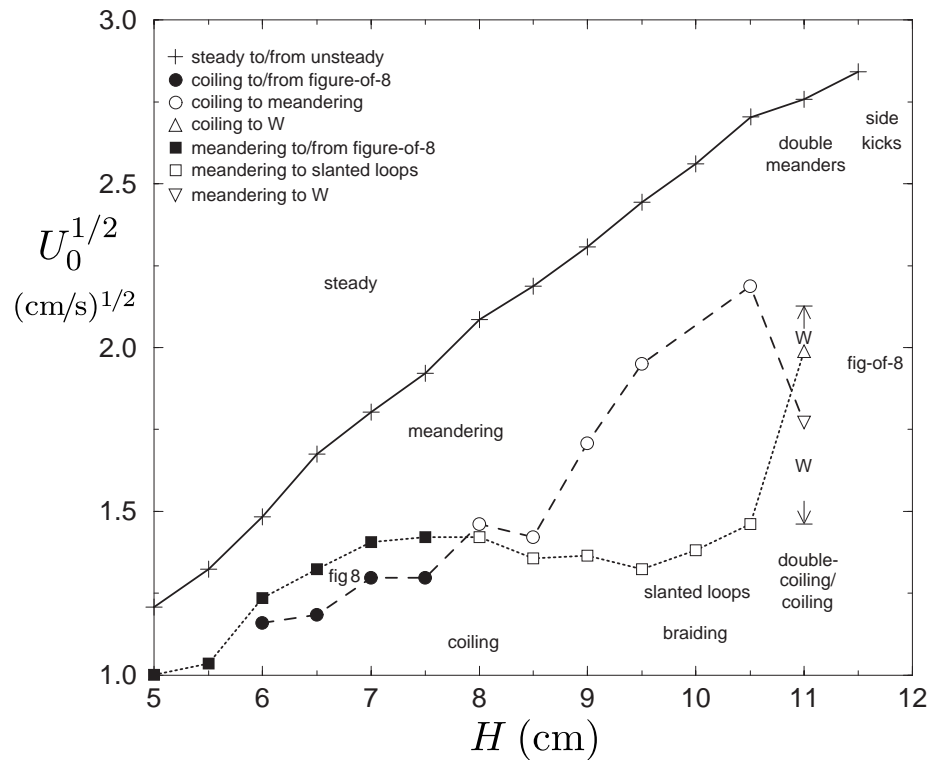


# Patterns from a Fluid-Mechanical Sewing Machine

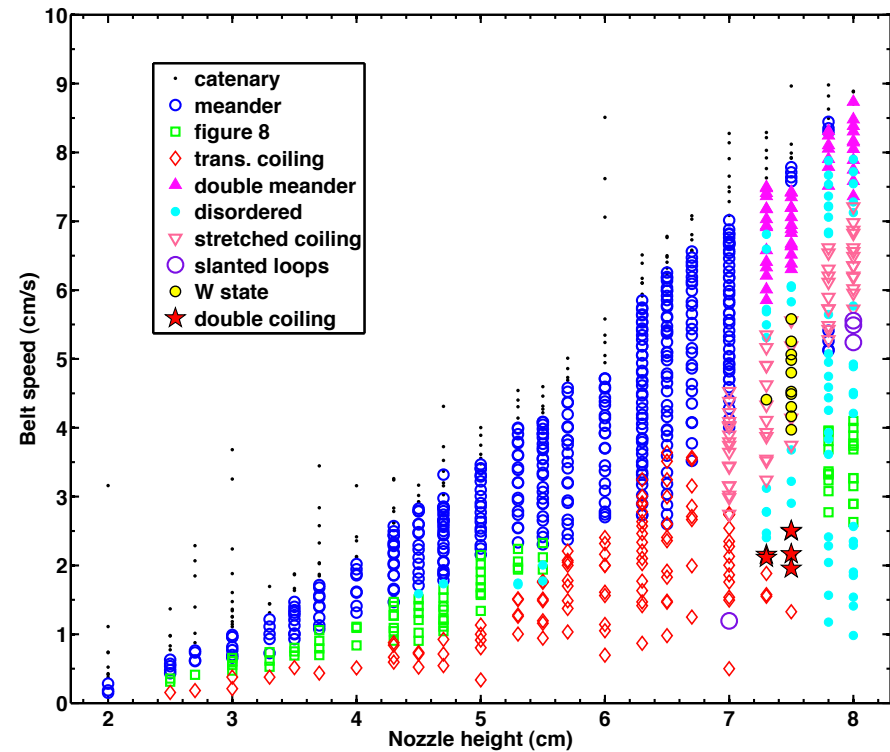


# Regime Diagrams

The control space  $(H, U_b)$  can be explored systematically



Chiu-Webster & Lister 2006

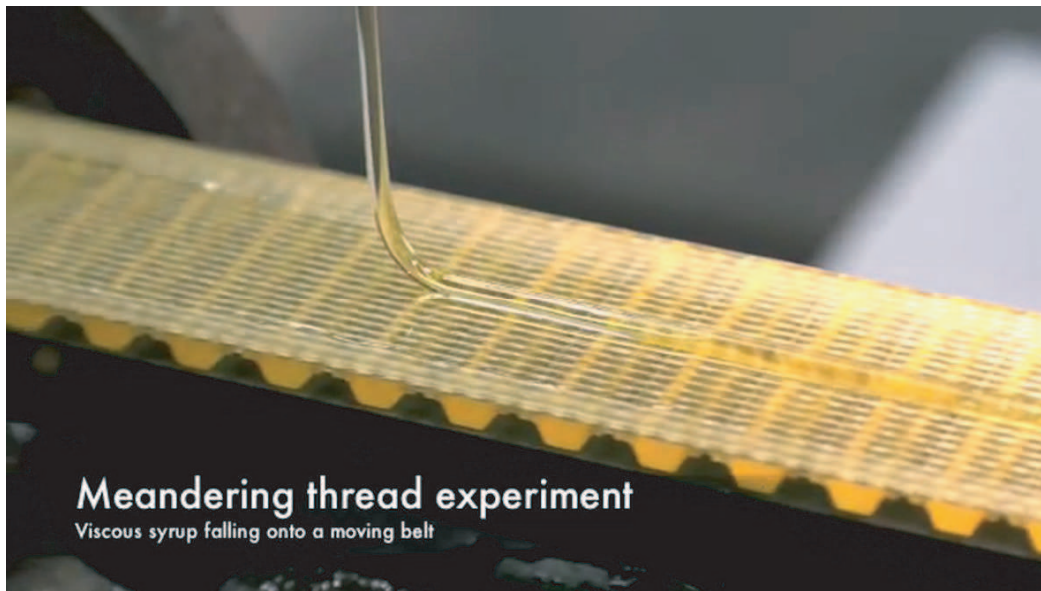


Morris et al. 2008

# Origins of Complexity

- Unsteadiness driven by overbalancing of the compressive heel
- Weak restoring by deflection of the tail
- also get resonances with pendulum modes in the tail
- also self-intersection on belt can lock/disrupt patterns

This is a beautiful system to show that the physics of viscous flow can be fun!

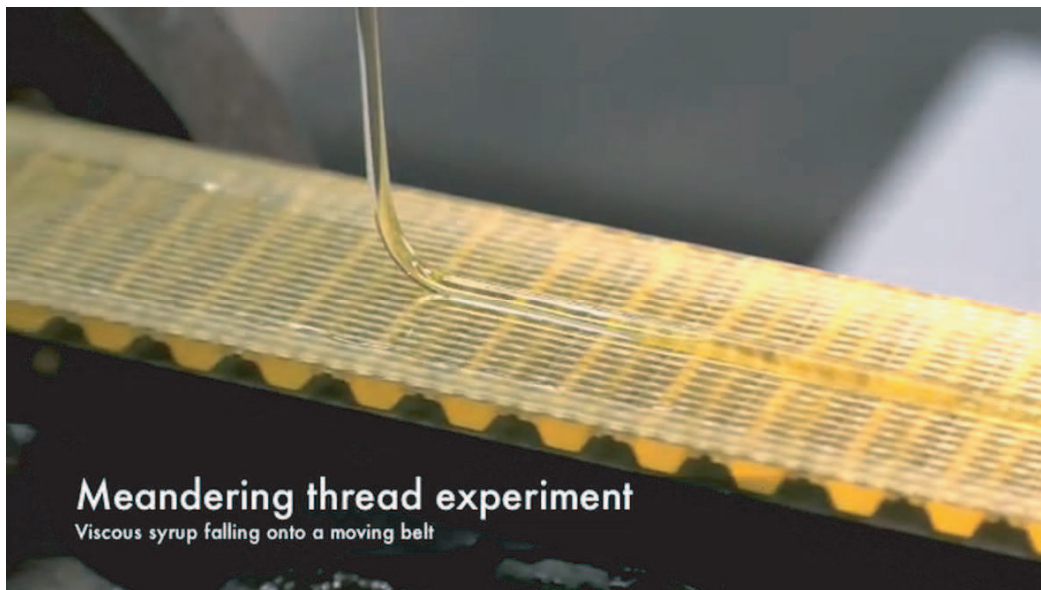


This movie by Stephen Morris is on U-Tube. The belt speed decreases slowly, keeping everything else constant.

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